



Adikari Wisesa
— **INDONESIA**



WEBINAR MAINTENANCE SERIES

PIPING VIBRATION INTEGRITY ASSESSMENT



Outline

First Session

- Introduction to Piping
- Piping Vibration Integrity Assessment (PVIA)
- PVIA Phase

Second Session

- Measurement Technique
- Case Study

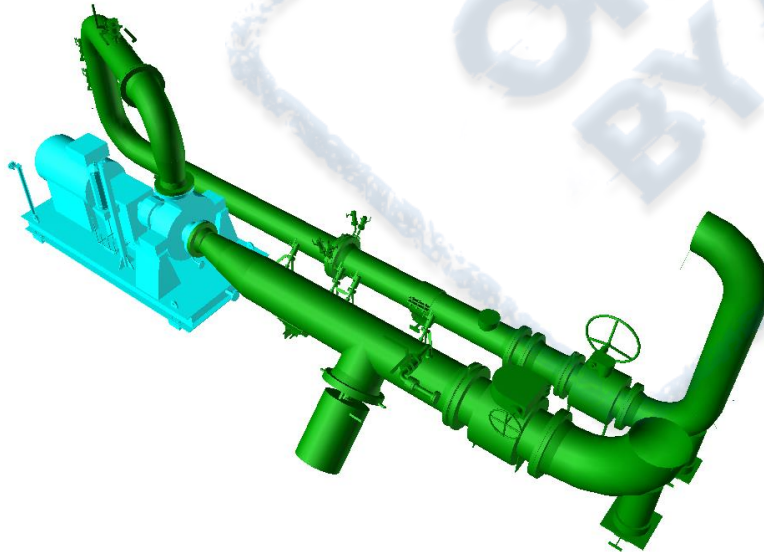
ORIGINAL
BY ADIKARI

First Session



Introduction to Piping

- *Piping is a system of pipes used to convey fluids (liquid and gases) from one location to another. The engineering discipline of piping design studies the efficient transport of fluid.*
- *Piping systems are like arteries and veins. They carry the lifeblood of modern civilization.*



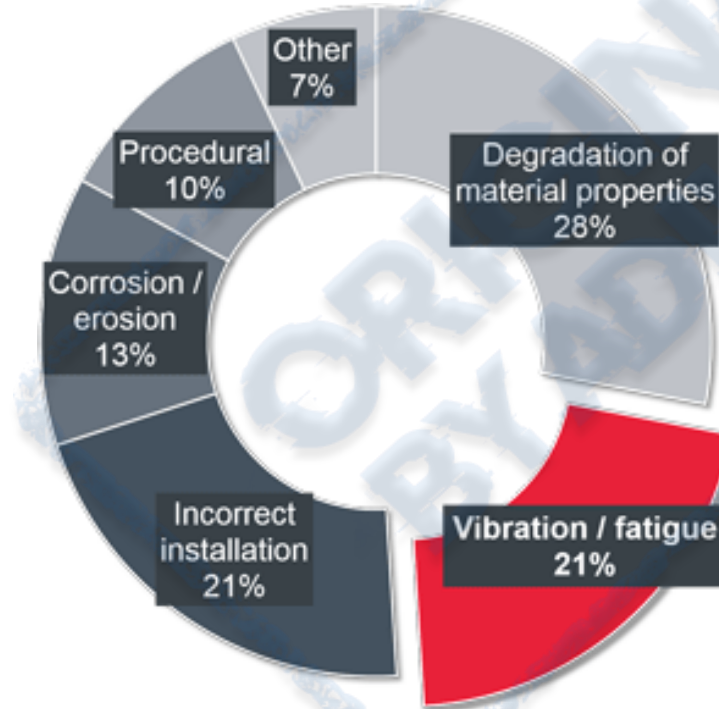
Introduction to Piping Vibration Integrity Assessment (PVIA)

What is vibration on piping?

Why does it happen?

What the effect?

How to avoid?



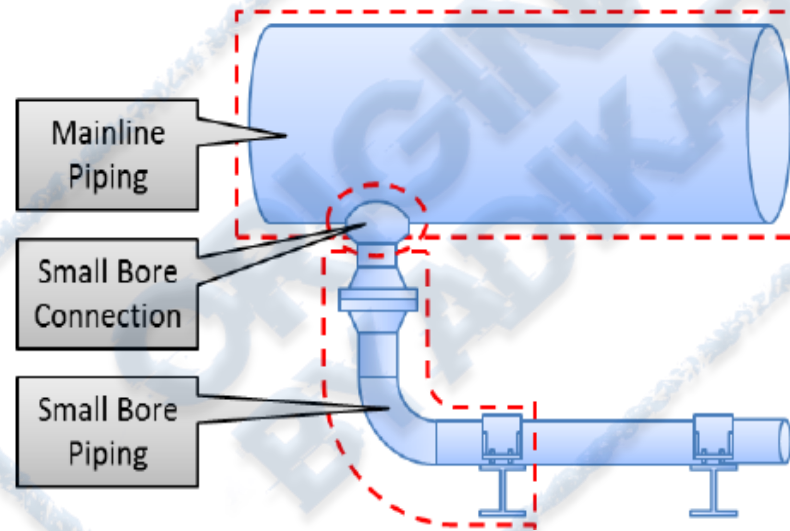
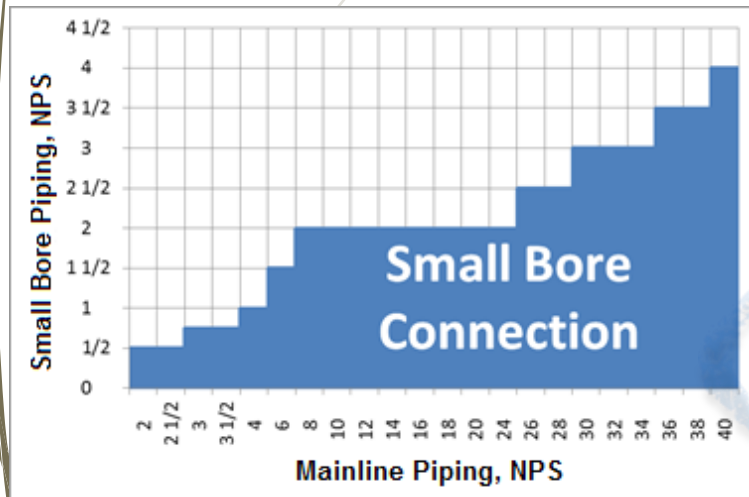
Pipe vibrates not only from inside/fluid but also propagates from others

- Other pipe
- Near equipment
- Deck/Support

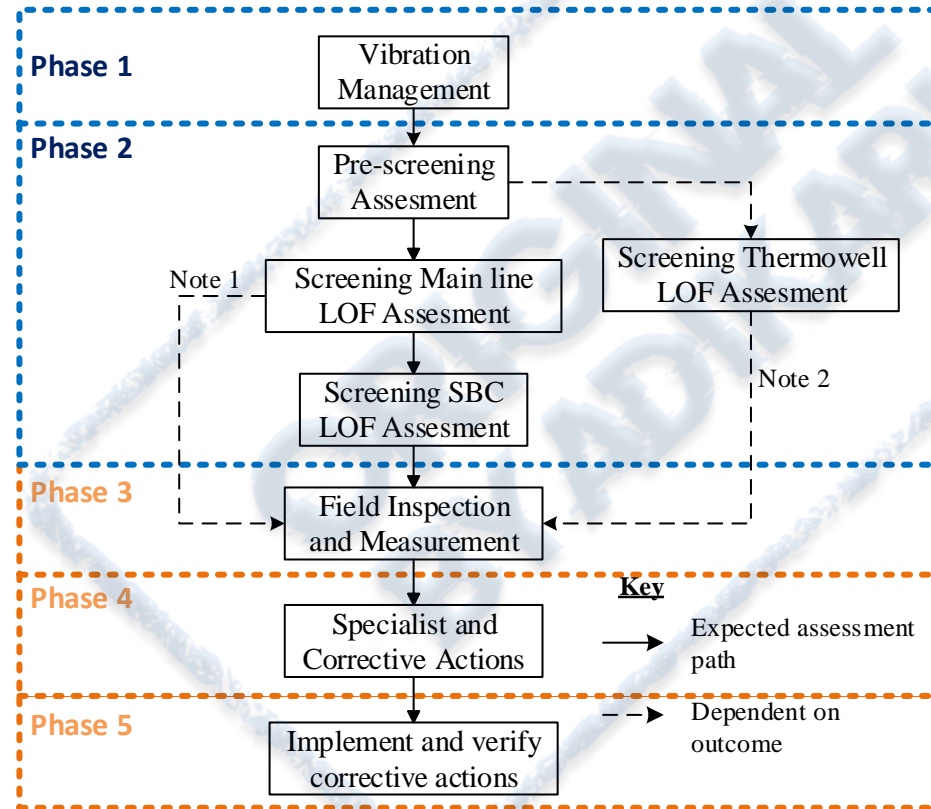
Can cause fatigue failure due to vibration
Avoid and mitigation?

- Diminish the cause
- Change layout of pipe or support
- Vibration monitoring

Introduction to Piping Vibration Integrity Assessment (PVIA)



PVIA Phase



Note 1 If LOF score greater than 0.5

Note 2 If LOF score is 1.0

Pre-Screening and Screening

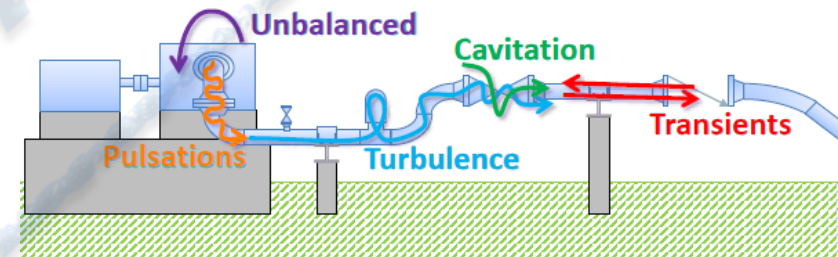
- Fluid Kinetic Energy
- Mechanical Excitation
- Pulsation
- High Frequency Acoustic
- Surge/Momentum
- Cavitation and Flashing

Pre-Screening

The goal is to identify potential LOF on the pipe

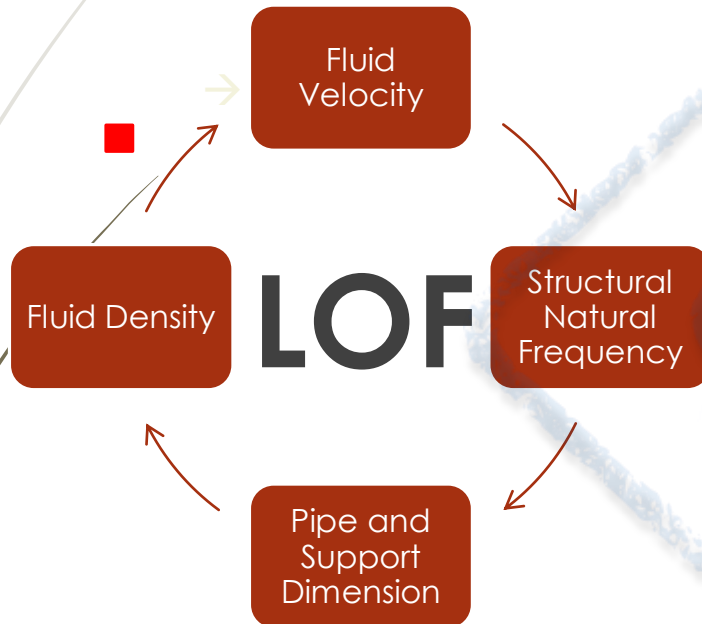
Screening

The goal is to quantitize the number of LOF on the pipe



Fluid Kinetic Energy

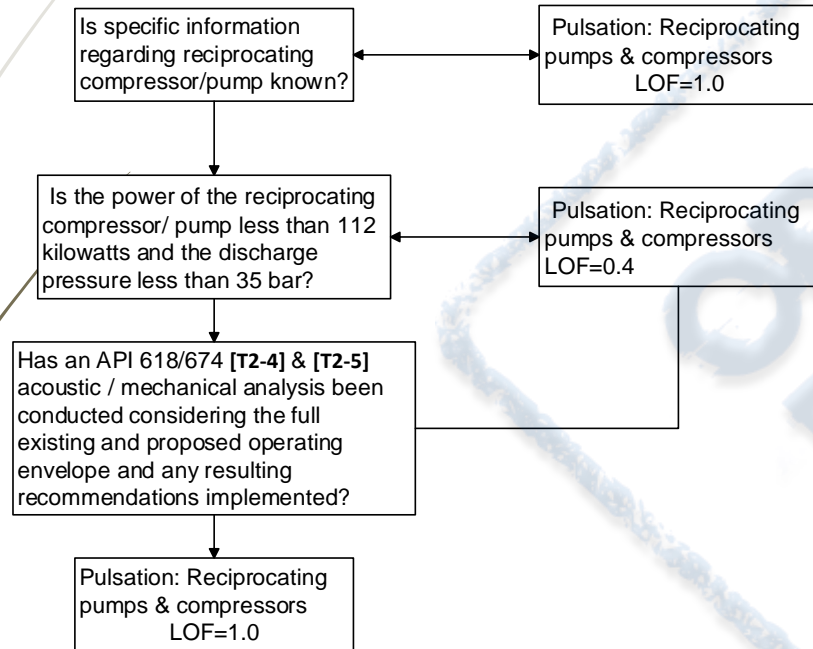
Mechanical Excitation



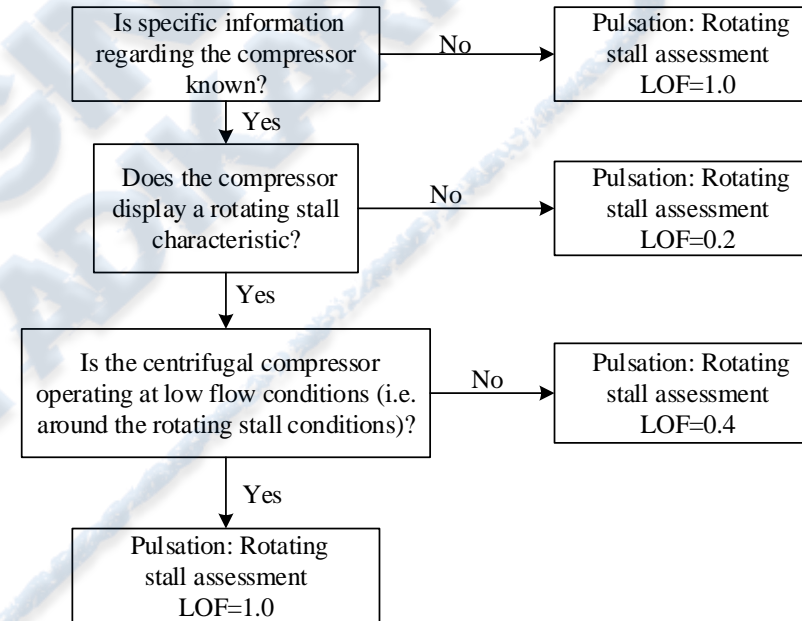
Pipework connected or adjacent to	LOF
Reciprocating/Positive Displacement Compressor/Pump	0.9
Diesel Engine / Gas Engine	0.8
Screw Compressor/Pump	0.6
Centrifugal Pump	0.4
Electric Motor/Alternator (15kW or greater)	0.4
Electric Motor/Alternator (below 15kW)	0.2
Centrifugal Compressor	0.2
Gas Turbine	0.2
Fan	0.2
Other pipework with an LOF ≥ 0.5	Equal to adjacent pipework LOF

Pulsation

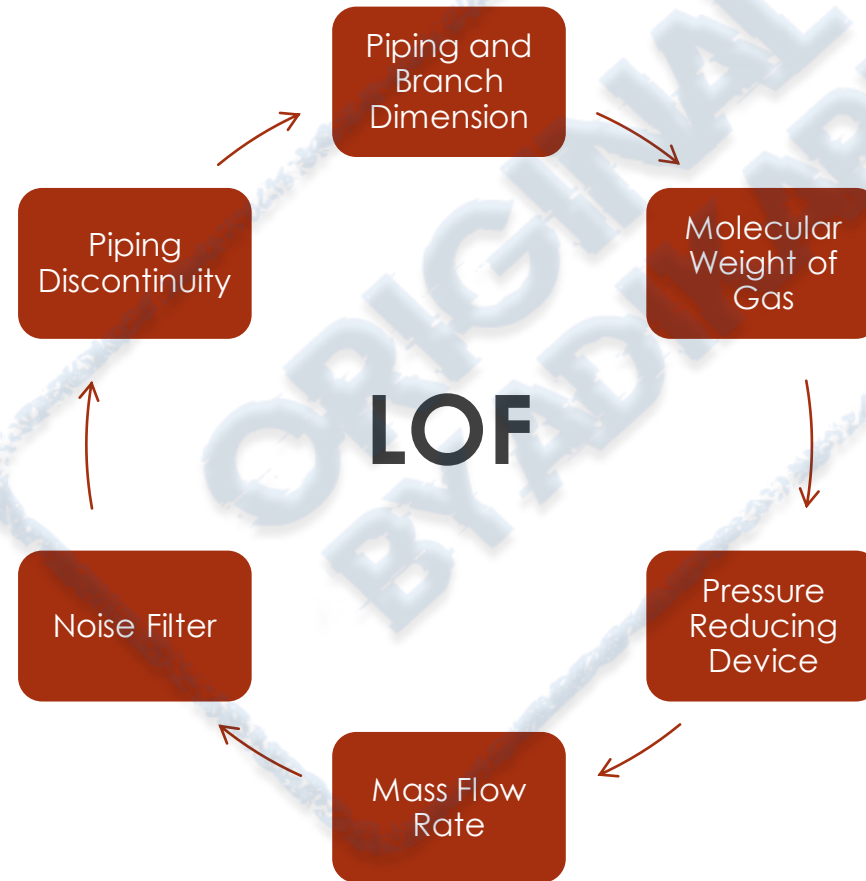
Reciprocating Equipment



Rotating Stall



High Frequency Acoustic



Surge/Momentum

Three Different Condition:

Dry Gas Valve Opening

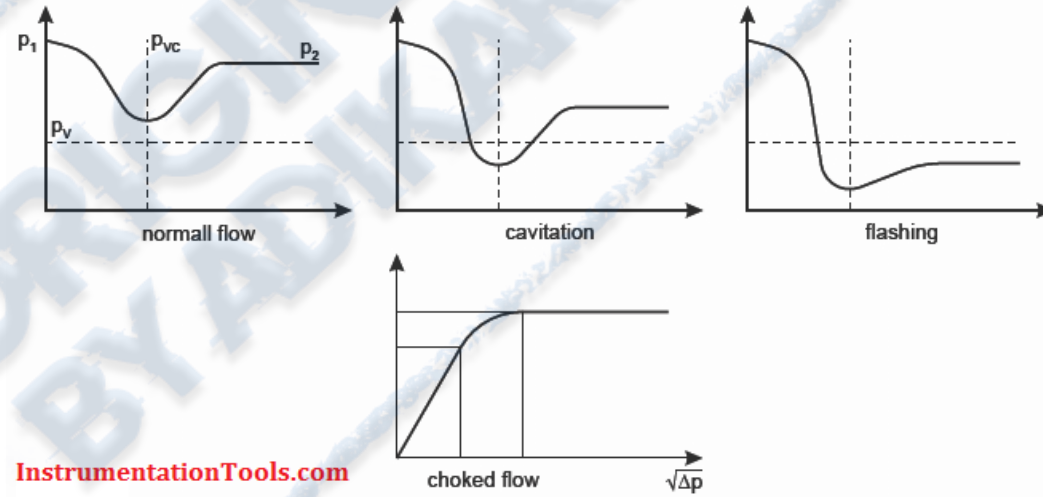
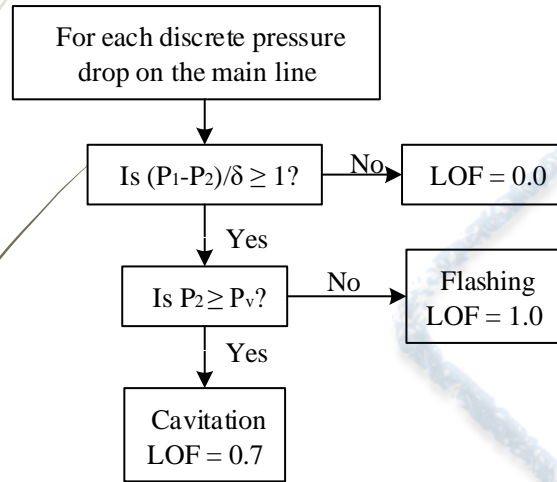
Liquid or Multiphase Valve Closure

Liquid or Multiphase Valve Opening

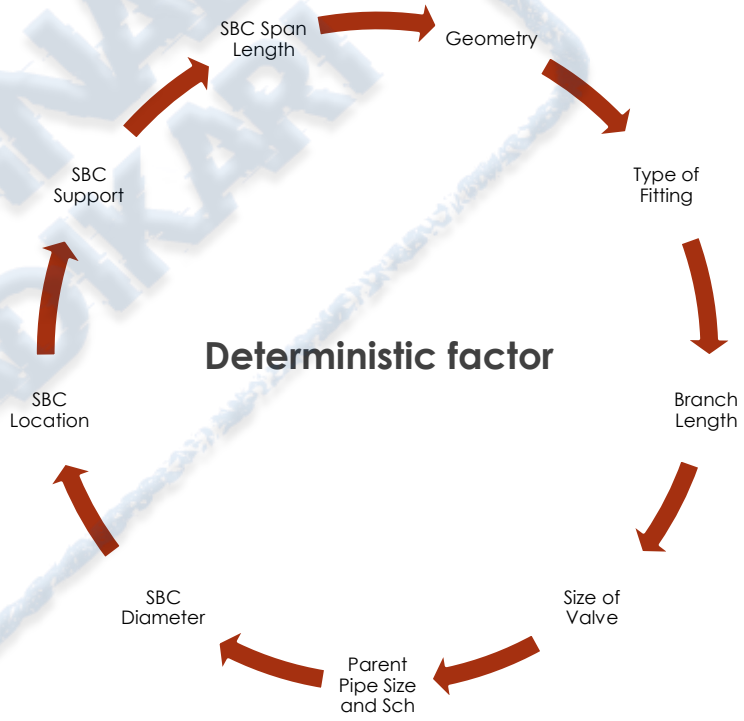
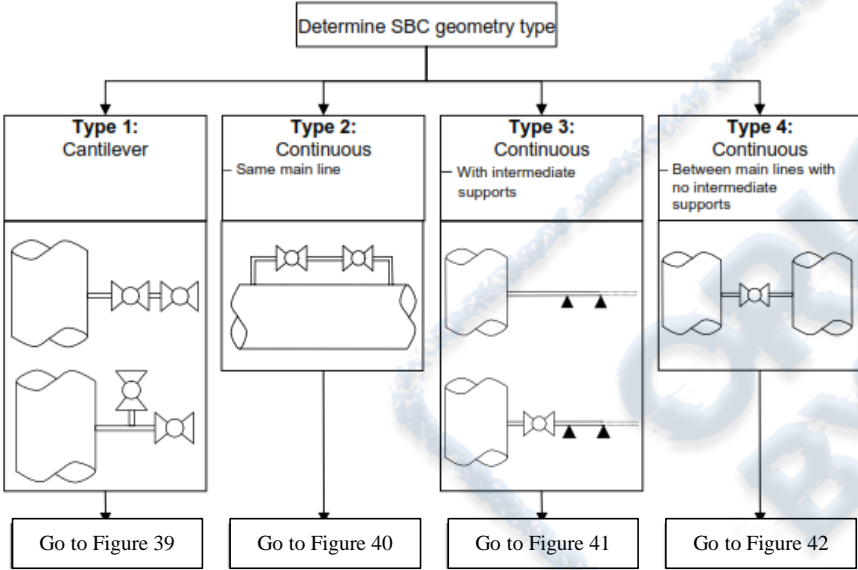
Properties

- Fluid properties (density, bulk modulus, molecular weight, specific heat capacity)
- Speed of sound
- Pipe properties (dimension, Young Modulus)
- Working properties (pressure, pressure drop, valve closing time, valve opening time, valve type)

Cavitation/Flashing



SBC Screening



Second Session



Measurement Techniques

Direct Strain Measurement



Direct assessment where fatigue failure is likely to happen

Fatigue as consideration for piping where crack is likely will happen.

Modal Analysis



To know vibration mode shape

Structural natural frequencies
Structural damping
Mode shape

Dynamic Pressure



Sensor capture fluid fluctuation

Very important data to see directly the flow behaviour

Vibration Measurement



To know vibration characteristics

By using vibration analyzer, the capture will be interpreted to see how the vibration occurs

Vibration Measurement

Mainline Vibration


Triaxial Axis on the weakest point or suspect to generate high vibration

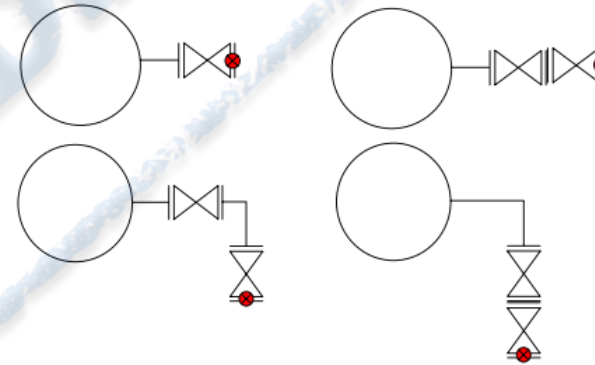
Common weak point on pipe :

- $\frac{1}{2}$ pipe span (between support)
- Long pipe span with external load (valve, tee, etc)
- Adjacent to vibrating deck/support
- Close to vibrating equipment
- Complex geometry without adequate support
- Before/after control valve

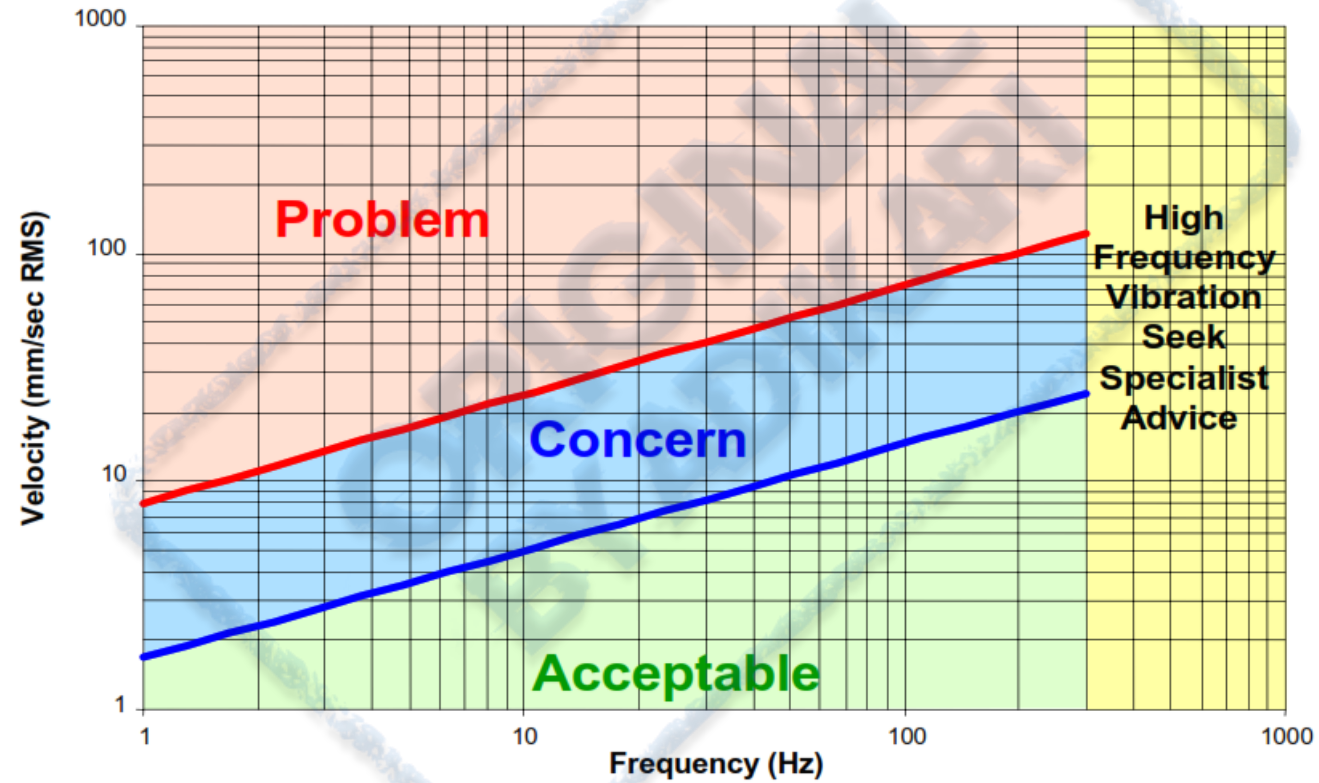
SBC Vibration

Triaxial Axis on the weakest point

Measurement Location -  (Note, measurements should be made in all three perpendicular directions)



Vibration Measurement



Paya Pasir, Medan

CASE STUDY

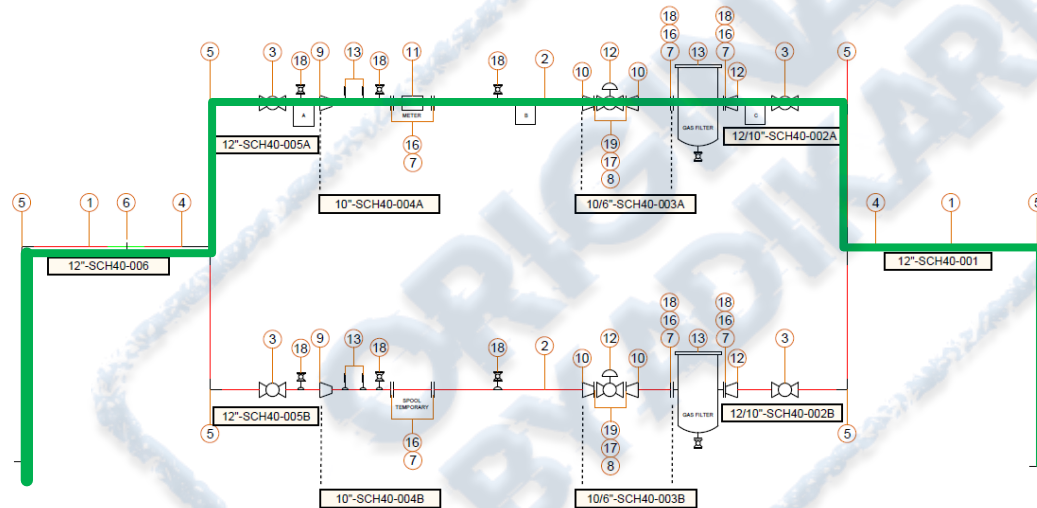
There was a gas metering station in Paya Pasir, Medan. After running day by day, week by week, month by month, a new symptom of noise started to emerge along with high vibration. In rural area, this noise propagated to other area. Client decided to find the root cause of the problem.

Overview and Scope of Work



- Overview
 - Noise exceed 85 dB and vibration emerge.
- Scope of Work
 - Design evaluation of qualitative and quantitative assessment.
 - Visual inspection and vibration measurement.
 - Data acquisition by special techniques.

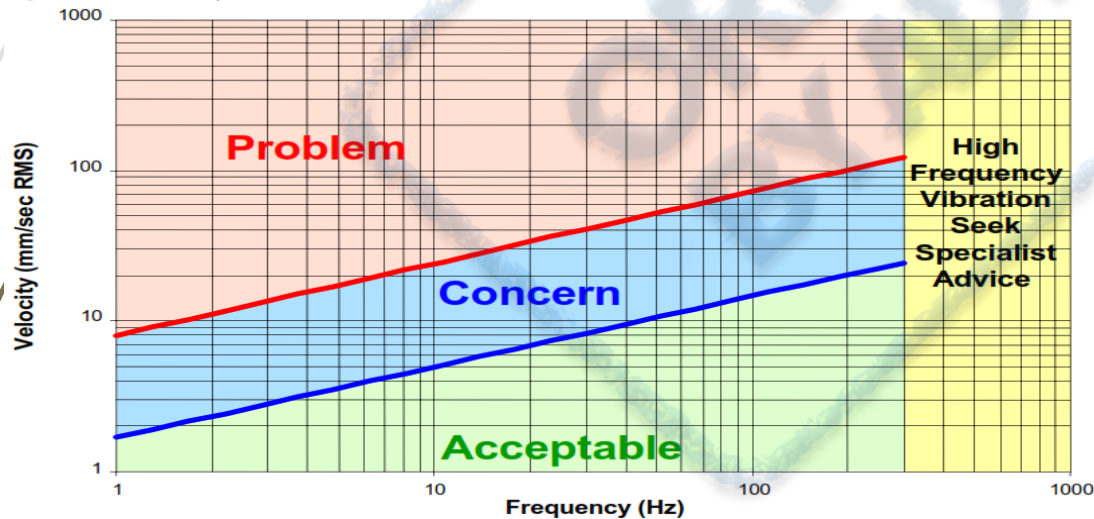
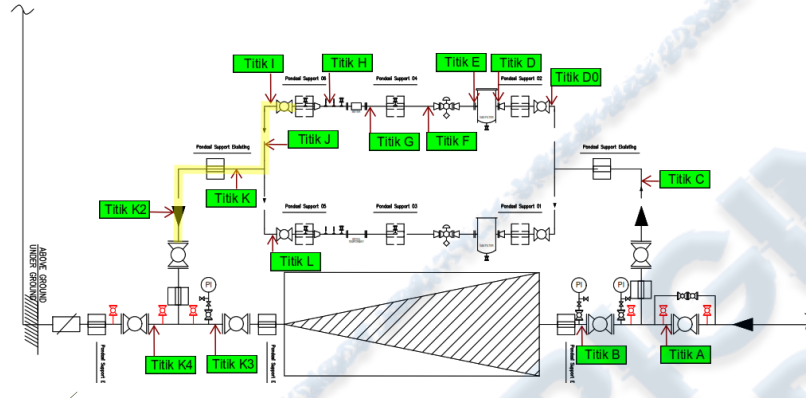
Screening Assessment



Aspek	Line 1	Line 2	Line 3	Line 4	Line 5	Line 6
Energi Kinetik	Low	Low	Medium	Low	Low	Low
Choke Flow	Low	High	High	High	High	Low
Machinery	Low	Low	Low	Low	Low	Low
Kompresor/P D Pumps	Low	Low	Low	Low	Low	High
Kavitasi/Flashi ng	Low	Low	Low	Low	Low	Low
Surge/Mome ntum	Low	High	High	Low	Low	Low

Line Number	Line List
Line 1	12"-SCH40-001
Line 2	12/10"-SCH40-002A
Line 3	10/6"-SCH40-003A
Line 4	10"-SCH40-004A
Line 5	12"-SCH40-005A
Line 6	12"-SCH40-006

Basic Vibration Assessment



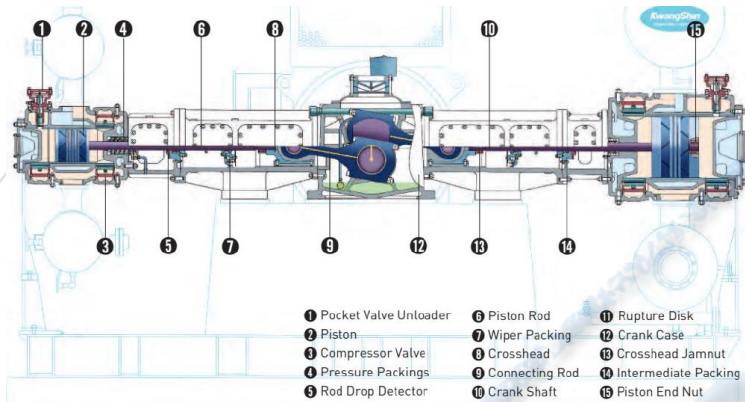
Nama Pipa	Axial		Vertikal		Horizontal		Kondisi
	Freq (Hz)	Amp mm/sec	Freq (Hz)	Amp mm/sec	Freq (Hz)	Amp mm/sec	
Titik I	50	0.4625	125.01	0.3443	49.98	1.0134	Minimum
	50.17	9.3072	50.16	8.0831	351.11	0.6497	Maksimum
Titik J	49.95	0.9133	49.97	0.6451	49.97	0.6451	Minimum
	49.94	2.1962	374.62	1.9804	1798.38	2.0511	Maksimum
Titik K	49.98	3.8425	50	3.3676	50	1.0436	Minimum
	49.91	10.7729	49.93	6.8217	49.95	1.8534	Maksimum
Titik L	25.01	0.1655	25.01	0.2774	50.01	1.3158	Minimum
	49.74	1.7772	99.45	1.3246	49.75	3.4206	Maksimum
Titik K2	49.93	0.7601	49.93	1.2219	49.94	2.2839	Minimum
	49.91	8.7317	49.94	5.0607	49.94	1.7133	Maksimum
Titik K3	49.9	1.2503	49.92	0.3166	24.97	0.6715	Minimum
	49.92	2.1637	99.84	0.6208	49.95	0.4676	Maksimum
Titik K4	50.02	0.5265	25.02	0.3554	25.01	0.5218	Minimum
	49.96	2.0632	49.96	0.8647	49.95	0.606	Maksimum

Basic Vibration Assessment



- Gas Booster Compressor
 - Gas Supply to GTG 3x25 MW
 - 700 m from Paya Pasir Metering Station
 - Pin: 15.5 bar and Pout: 34 bar
 - 5 unit Booster Compressor
 - Driven by electric motor 280 kW with 1485 rpm.

Basic Vibration Assessment

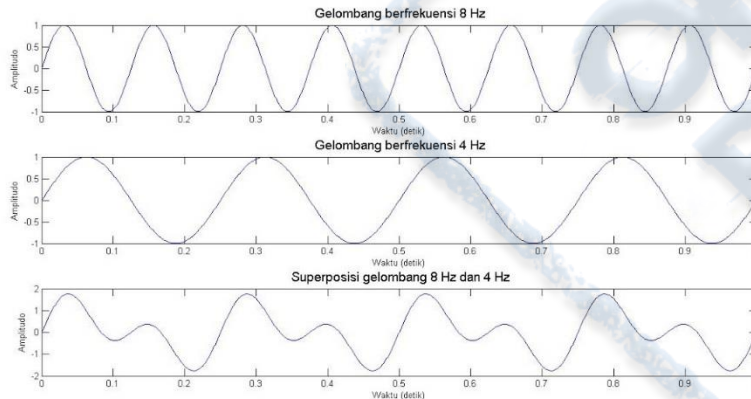


Axial			Vertikal			Horizontal		
PEAK	FREQUENC	PEAK	PEAK	FREQUENC	PEAK	PEAK	FREQUENC	PEAK
NO.	(Hz)	VALUE	NO.	(Hz)	VALUE	NO.	(Hz)	VALUE
1	24.97	10.4615	1	24.95	4.8827	1	24.95	12.7153
2	49.94	7.73	2	49.91	3.5562	2	49.91	3.6263
3	74.92	6.4753	3	74.87	18.9253	3	74.86	5.5868
4	99.89	2.3789	4	99.82	2.5076	4	99.83	2.0514
5	124.87	2.2347	5	124.78	2.4512	5	124.78	4.1703

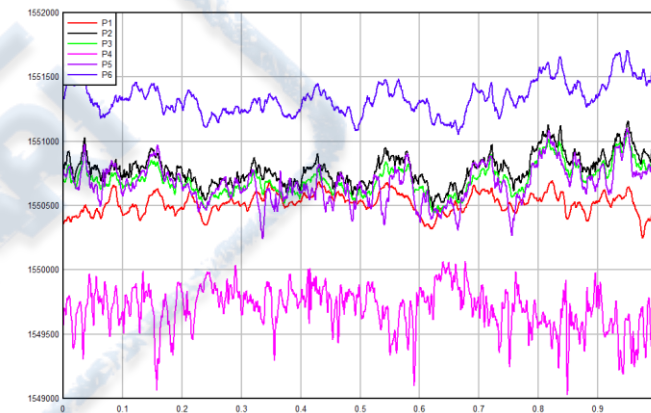
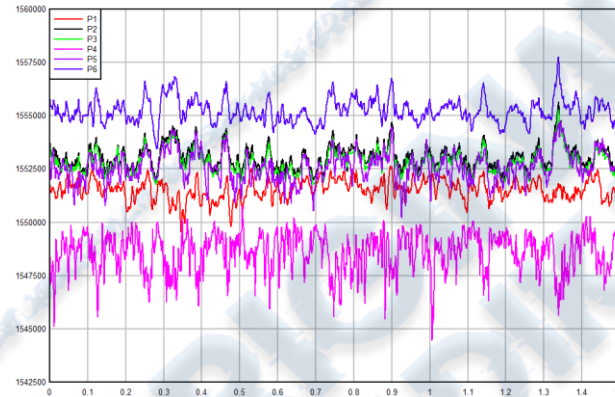
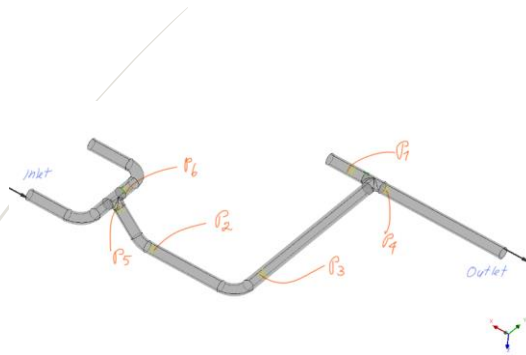
➤ Basic Vibration Measurement

Electric motor operates at frequency 25 Hz (1485 RPM)

- Double acting reciprocating compressor cause the excitation source at 50 Hz.
- Harmonic frequency of driver running speed 25 Hz (25, 50, 75 Hz,...etc) propagates to Paya Pasir through fluid or pipeline.
- Compressor configuration cause excitation source propagates to Paya Pasir up and down depends on superposition.



Computational Fluid Dynamics



Titik	Model A (kiri)		Model B (kanan)	
	Max ΔP (frekuensi)	ΔP pd 50Hz	Max ΔP (frekuensi)	ΔP pd 50Hz
P1	170 Pa (<5 Hz)	30 Pa	90 Pa (8 Hz)	3 Pa
P2	224 Pa (< 5 Hz)	20 Pa	45 Pa (< 5 Hz)	5 Pa
P3	210 Pa (< 5 Hz)	20 Pa	90 Pa (< 5 Hz)	5 Pa
P4	226 Pa (15 Hz)	50 Pa	85 Pa (< 5 Hz)	25 Pa
P5	290 Pa (< 5 Hz)	60 Pa	99 Pa (< 5 Hz)	10 Pa
P6	180 Pa (< 5 Hz)	50 Pa	99 Pa (< 5 Hz)	5 Pa

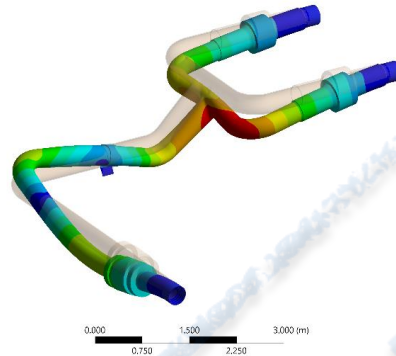
➤ Computational Fluid Dynamics Result

- Pressure fluctuation inside the pipe happens.
- Pictures above in terms of waveform that can interpret in FFT.
- There are pressure fluctuation in 50 Hz.
- Maximum pressure fluctuation in low frequency (<5 Hz).
- Maximum pressure fluctuation at 290 Pa (operate at 1500 kPa).

Modal Analysis

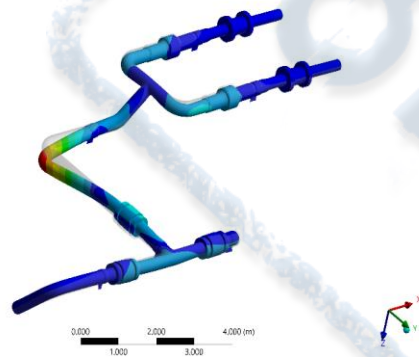
J: Modal
 Total Deformation
 Type: Total Deformation
 Frequency: 49.013 Hz
 Unit: m
 7/14/2019 8:41 PM

0.04264 Max
 0.037902
 0.033154
 0.028427
 0.023689
 0.018951
 0.014213
 0.0094756
 0.0047378
0 Min



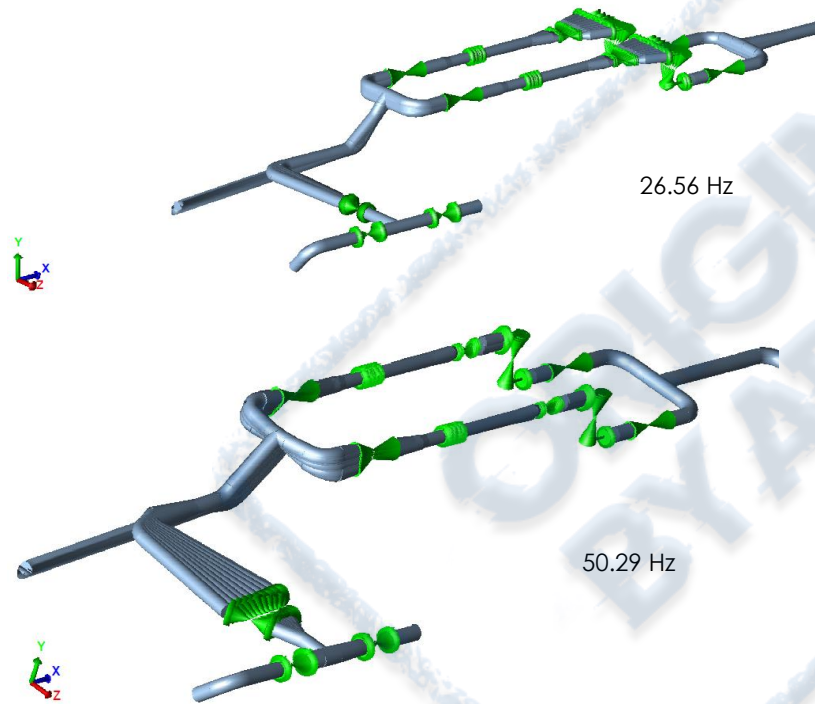
G: Copy of Modal
 Total Deformation
 Type: Total Deformation
 Frequency: 45.348 Hz
 Unit: m
 7/14/2019 9:18 PM

0.068084 Max
 0.060519
 0.052955
 0.04539
 0.037825
 0.03026
 0.022695
 0.01513
 0.0075649
0 Min



Mode	Model A (atas)	Model B (bawah)
1	29.05	13.75
2	41.69	18.98
3	44.08	23.90
4	49.01	28.45
5	59.76	29.30
6	63.10	31.80
7	85.14	33.15
8	113.40	34.99
9	122.58	36.00
10	125.74	49.33

Modal Analysis



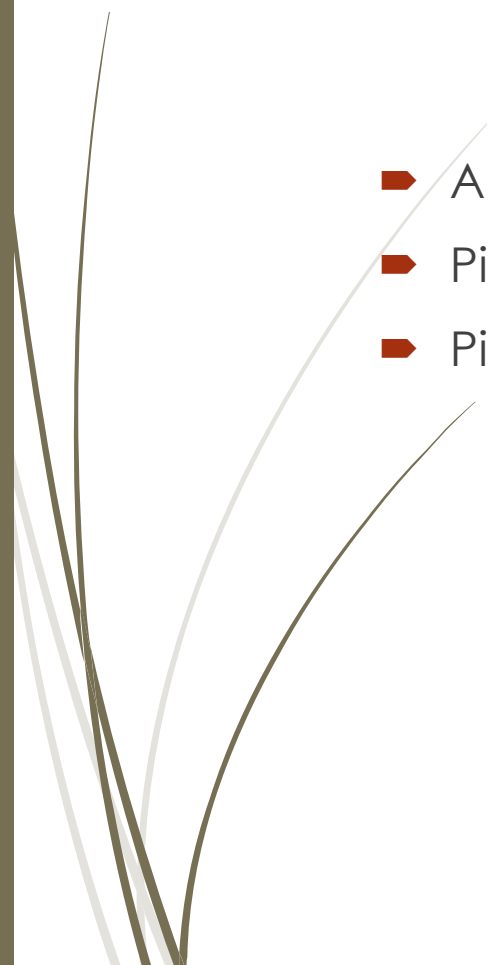

Mode	Frequency (Hz)
1	3.169
2	15.246
3	20.314
4	20.803
5	26.56
6	27.91
7	29.83
8	30.21
9	35.56
10	36.18
11	36.24
12	36.24
13	38.51
14	39.25
15	40.53
16	42.13
17	43.52
18	48.86
19	50.29
20	52.91
21	57.6
22	59.647
23	60

Conclusion

- From assessment screening, potential excitation are kinetic energy, choke flow, compressor pulsation, surge/momentum.
- Vibration measurement in Paya Pasir shows maximum overall vibration level of 10.77 mm/sec at Point K (concern) with dominant frequency 50 Hz.
- Configuration of gas booster compressor cause wave superposition.
- CFD result shows pressure fluctuation with maximum 290 Pa (operation condition 1500 kPa). Low pressure fluctuation at 50 Hz.
- Vibration at Paya Pasir Metering Station caused by :
 - Excitation at 50 Hz. Excitation sources comes from pulsation of gas booster compressor.
 - Acoustic natural frequency and mechanical natural frequency at 50 Hz.



Recommendation

- Add dampener to minimize pulsation effect. Sizing needed.
 - Piping route modification to shift the mechanical natural frequency.
 - Piping support modification to shift the mechanical natural frequency.
- 
- 

THANK YOU

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extreme-maintenance.com